

**ALERT Geomaterials Doctoral School 2018**

# **The Principle of Virtual Powers definition and use**

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# **Objectives**

- Understand the Principle of Virtual Powers
- Use it in simple problems of statics
- Use it in continuum mechanics
- Use it as a starting point for Finite Element formulations

# **Prerequisites**

- **Tensor calculus**
- Patience and a nice problem to solve

# The Principle of Virtual Powers



*Giuseppe Luigi Lagrangia, 1736-1813 (Joseph-Louis Lagrange)*

# MÉCHANIQUE

## ANALITIQUE;

Par M. DE LA GRANGE, de l'Académie des Sciences de Paris,

de celles de Berlin, de Pétersbourg, de Turin, &c.



#### $A$   $P$   $A$   $R$   $I$   $S$ ,

Chez LA VEUVE DESAINT, Libraire, rue du Foin S. Jacques.





# MÉCHANIQUE ANALITIQUE.

PREMIERE PARTIE

LA STATIQUE.

#### SECTION PREMIERE.

Sur les différens Principes de la Statique.

LA Statique eft la fcience de l'équilibre des forces. On entend en général par force ou puiffance la caufe, quelle qu'elle foit, qui imprime ou tend à imprimer du mouvement au corps auquel on la fuppofe appliquée; & c'eft auffi par la quantité du mouvement imprimé, ou prêt à imprimer, que la force ou puiffance doit s'eftimer.



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- **Statics** is the science of the **equilibrium of forces**
- With *force* (or *power*) we mean the *cause of mouvement*
- It is through the quantity of mouvement that the force can be quantified

# **The Principle of Virtual Velocities (or Powers)**

Si un fystême quelconque de tant de corps ou points que L'on veut tirés, chacun par des puissances quelconques, est en équilibre, & qu'on donne à ce fystême un petit mouvement quelconque, en vertu duquel chaque point parcoure un efpace infiniment petit qui exprimera fa viteffe virtuelle; la fomme des puiffances, multipliées chacune par l'espace que le point où elle est appliquée, parcourt fuivant la direction de cette même puiffance, fera toujours égale à zero, en regardant comme positifs les petits espaces parcourus dans le sens des puissances, & comme négatifs les espaces parcourus dans un fens oppofé.

# **Original statement of PVP in English**

**If** [any system of bodies or points as we want, is acted upon by any system of forces,] **is in equilibrium**, **and we give to this system any small motion, then** [by virtue of the fact that each point travels an infinitesimally small space that expresses its **virtual velocity**,] **the sum of each force multiplied by the space that the point** [where it is applied] **travels** [along the direction of the same power,] **will always be equal to zero**, [regarding as positive the small distances followed in the

direction of the powers and as negative those travelled in the opposite direction.]

**The body will be in equilibrium** 

*if, and only if,* 

**the power generated by the forces acting on it, is zero** under **any possible (= virtual) velocity** of the body.

Lagrange was far ahead of his time...

A long time ago, in a galaxy far, far away....





# **The scientific problem:**



# **PVP applied to a single body**



equilibrium  $\iff \sum_i \widetilde{\mathcal{P}}^{(i)} = 0, \forall \widetilde{\mathbf{v}}^{(i)}$  $\sum_i \boldsymbol{F}^{(i)} \cdot \widetilde{\boldsymbol{v}}^{(i)} = 0, \forall \widetilde{\boldsymbol{v}}^{(i)}$ Rigid body:  $\widetilde{\bm{v}}^{(i)} = \widetilde{\bm{v}}^O + \widetilde{\bm{\omega}}^O \times \bm{r}^{(O,i)}$  $\left(\sum_i \boldsymbol{F}^{(i)}\right) \cdot \widetilde{\boldsymbol{v}}^O + \left(\sum_i \boldsymbol{r}^{(O,i)} \times \boldsymbol{F}^{(i)}\right) \cdot \widetilde{\boldsymbol{\omega}}^O = 0, \forall \widetilde{\boldsymbol{v}}^O, \widetilde{\boldsymbol{\omega}}^O$  $\sum_i \boldsymbol{F}^{(i)} = 0$  $\forall \widetilde{v}^O, \widetilde{\omega}^O \implies$  $\sum_i \boldsymbol{M}^{(O,i)} = 0$ 

# **PVP and 'Newton' equivalence**

equilibrium 
$$
\iff
$$
  $\sum_{i} \mathbf{F}^{(i)} = 0$   
 $\sum_{i} \mathbf{M}^{(O,i)} = 0$ 

multiplying by any  $\tilde{v}^{\scriptscriptstyle O},\tilde{\omega}^{\scriptscriptstyle O}$  and adding:

$$
\left(\textstyle\sum_i \boldsymbol{F}^{(i)}\right)\cdot \widetilde{\boldsymbol{v}}^O + \left(\textstyle\sum_i \boldsymbol{r}^{(O,i)}\times\boldsymbol{F}^{(i)}\right)\cdot \widetilde{\boldsymbol{\omega}}^O=0, \forall \widetilde{\boldsymbol{v}}^O, \widetilde{\boldsymbol{\omega}}^O
$$

Rigid body:  $\widetilde{v}^{(i)} = \widetilde{v}^O + \widetilde{\omega}^O \times r^{(O,i)}$ 

$$
\sum_i \boldsymbol{F}^{(i)} \cdot \widetilde{\boldsymbol{v}}^{(i)} = 0, \forall \widetilde{\boldsymbol{v}}^{(i)} \iff \left| \sum_i \widetilde{\mathcal{P}}^{(i)} = 0, \forall \widetilde{\boldsymbol{v}}^{(i)} \right|
$$



# **Exercise #1**

-> *Using the PPV find all the equilibrium points (angles θ) of the system*:



## **Answer #1**



# **PVP applied to a system of bodies**





$$
\sum_{i} \widetilde{\mathcal{P}}^{(i)} = \sum_{j} \widetilde{\mathcal{P}}^{(ext,j)} - \sum_{k} \widetilde{\mathcal{P}}^{(int,k)} = 0, \qquad \forall \widetilde{\mathbf{v}}^{(i)}
$$
  
-  

$$
\widetilde{\delta}_{D} = \widetilde{\alpha} \frac{L}{4} = \widetilde{\beta} \frac{3L}{4}
$$
  

$$
\widetilde{\delta}_{C} = \widetilde{\beta} \frac{L}{2}
$$

# **Exercise #2**

-> *Using the PPV find the internal moment at point C:*



## **Answer #2**

# $M_D^{int}=\frac{FL}{4}$







# Continuum Mechanics

# **PVP and continuum mechanics**





#### Definitions

internal deformation enery density:  $\widetilde{p}^{(int)}=\sigma_{ij}\widetilde{\varepsilon}_{ij}$ 

$$
\widetilde{\varepsilon}_{ij} = \frac{1}{2} \left( \widetilde{v}_{i,j} + \widetilde{v}_{j,i} \right), \quad \sigma_{ij} = \sigma_{ji}
$$

external enery density:  $\widetilde{p}^{(ext,t)} = t_i \widetilde{v}_i$ 

$$
\widetilde{p}^{(ext,f)}=f_i\widetilde{v}_i
$$

$$
\int_{V} \sigma_{ij} \widetilde{\varepsilon}_{ij} dV - \int_{V} f_{i} \widetilde{v}_{i} dV - \int_{\partial V} t_{i} \widetilde{v}_{i} dS = 0, \forall \widetilde{v}_{i}
$$

$$
\int_{V} \sigma_{ij} \widetilde{\varepsilon}_{ij} dV = \int_{V} \sigma_{ij} \widetilde{v}_{i,j} dV = \int_{V} (\sigma_{ij} \widetilde{v}_{i})_{,j} dV - \int_{V} \sigma_{ij,j} \widetilde{v}_{i} dV
$$

Using the divergence theorem:

$$
\int_{V} (\sigma_{ij} \widetilde{v}_i)_{,j} dV = \int_{\partial V} (\sigma_{ij} \widetilde{v}_i) n_j dS = \int_{\partial V} (\sigma_{ij} n_j) \widetilde{v}_i dS
$$

Replacing:  $\int_V (\sigma_{ij,j} + f_i) \widetilde{v}_i dV + \int_{\partial V} (t_i - \sigma_{ij} n_j) \widetilde{v}_i dS = 0, \forall \widetilde{v}_i$ 

$$
\int_{V} \left(\sigma_{ij,j} + f_i\right) \widetilde{v}_i dV + \int_{\partial V} \left(t_i - \sigma_{ij} n_j\right) \widetilde{v}_i dS = 0, \forall \widetilde{v}_i
$$

$$
\iff \begin{cases} \sigma_{ij,j} + f_i = 0 \\ t_i = \sigma_{ij} n_j \end{cases}
$$

Which are the classical equilibrium equations in continuum mechanics -> Strong form



# Linear & angular momentum balance principles

• 
$$
\int_{\partial V} t_i dS + \int_V f_i dV = 0
$$
  
\nwith  $t_i = \sigma_{ij} n_j$   
\n•  $\int_{\partial V} \epsilon_{ijk} t_j x_k dS + \int_{\partial V} \epsilon_{ijk} f_j x_k dV = 0$   
\n  
\n  
\n
$$
\int_{\partial V} t dS + \int_V f dV = \int_{\partial V} \vec{t} dS + \int_V \vec{f} dV
$$
  
\n
$$
\int_{\partial V} t \times x dS = \int_{\partial V} \vec{t} \times \vec{x} dS
$$

$$
\int_{\partial V} t_i dS + \int_V f_i dV = 0
$$

 $t_i = \sigma_{ij} n_j$ 

Using the divergence theorem:

$$
\int_{\partial V} t_i dS = \int_{\partial V} \sigma_{ij} n_j dS = \int_V \sigma_{ij,j} dV
$$

Replacing:

$$
\int_{V} (\sigma_{ij,j} + f_i)dV = 0
$$

$$
\int_{\partial V} \epsilon_{ijk} t_j x_k dS + \int_V \epsilon_{ijk} f_j x_k dV = 0
$$

 $t_i = \sigma_{ij} n_j$ 

Using the divergence theorem:

$$
\int_{\partial V} \epsilon_{ijk} t_j x_k dS = \int_{\partial V} \epsilon_{ijk} \sigma_{jp} x_k n_p dS = \int_V (\epsilon_{ijk} \sigma_{jp} x_k)_{,p} dV =
$$

$$
= \int_V (\epsilon_{ijk}\sigma_{jp}x_k)_{,p} dV = \int_V \epsilon_{ijk}\sigma_{jp,p}x_k dV + \int_V \epsilon_{ijk}\sigma_{jp}x_{k,p} dV =
$$

$$
= \int_{V} \epsilon_{ijk} (-f_j) x_k dV + \int_{V} \epsilon_{ijk} \sigma_{jp} \delta_{kp} dV
$$

$$
= -\int_{V} \epsilon_{ijk} f_{j} x_{k} dV + \int_{V} \epsilon_{ijk} \sigma_{jk} dV
$$

# Replacing:

$$
\int_{\partial V} \epsilon_{ijk} t_j x_k dS + \int_V \epsilon_{ijk} f_j x_k dV = \int_V \epsilon_{ijk} \sigma_{jk} dV = 0
$$

$$
\int_{V} \epsilon_{ijk} \sigma_{jk} dV = 0
$$

$$
\sigma_{jk}=\sigma_{kj}
$$

Starting point:

- Virtual (generalized) displacements
- Virtual powers (external & internal)

Less intuition

Prone to the Galerkin Method (Finite Elements)

### PVP Momentum Balance

Starting point:

• (Generalized) stresses

More intuition

# **Principle of Virtual Powers**

**or** 

# **Principle linear & angular momentum balance ?**

# **It is a matter of principles!**

# **Cosserat Continuum**

**Lecture Notes in Applied and Computational Mechanics 87** 

#### **Ioannis Vardoulakis**

# Cosserat Continuum Mechanics

**With Applications to Granular Media** 



# **From the MB point of view**





## Equilibrium on infinitesimal volume & Cauchy tetrahedron:

$$
\tau_{ij,j} + f_i = 0, \qquad t_i = \tau_{ij} n_j
$$

$$
m_{ij,j} - \epsilon_{ijk} \tau_{jk} + \psi_i = 0, \qquad \mu_i = m_{ij} n_j
$$

# **Energetic approach: PVP**



$$
\widetilde{p}^{(ext,f)} = f_i \widetilde{v}_i + \psi_i \widetilde{\omega}_i^c
$$
\n
$$
\widetilde{p}^{(ext,t)} = t_i \widetilde{v}_i + \mu_i \widetilde{\omega}_i^c
$$
\n
$$
\widetilde{p}^{int} = \tau_{ij} \widetilde{\gamma}_{ij} + m_{ij} \widetilde{k}_{ij}
$$
\n
$$
\gamma_{ij} = u_{i,j} + \epsilon_{ijk} \omega_k^c
$$
\n
$$
k_{ij} = \omega_{i,j}^c
$$

# **Exercise #3**

-> *Retrieve the strong form of equilibrium equations for Cosserat:*

$$
\widetilde{p}^{(ext,f)} = f_i \widetilde{v}_i + \psi_i \widetilde{\omega}_i^c
$$
\n
$$
\widetilde{p}^{(ext,t)} = t_i \widetilde{v}_i + \mu_i \widetilde{\omega}_i^c
$$
\n
$$
\widetilde{p}^{int} = \tau_{ij} \widetilde{\gamma}_{ij} + m_{ij} \widetilde{k}_{ij}
$$
\n
$$
\gamma_{ij} = u_{i,j} + \epsilon_{ijk} \omega_k^c
$$
\n
$$
k_{ij} = \omega_{i,j}^c
$$

$$
\tau_{ij,j} + f_i = 0
$$
  
\n
$$
m_{ij,j} - \epsilon_{ijk}\tau_{jk} + \psi_i = 0
$$
  
\n
$$
t_i = \tau_{ij}n_j
$$
  
\n
$$
\mu_i = m_{ij}n_j
$$

# **1D Cosserat: The Timoshenko beam**

Consider a narrow infinite strip of Cosserat continuum in the *x*<sub>2</sub>-direction, invariant in the *x*1 - and *x*<sup>3</sup> -directions:



 $X_2$ 

$$
V \equiv \tau_{32} A_c
$$
  
Call:  $M \equiv -m_{12} A_c$   $\longrightarrow$   $\frac{dV}{dx} = 0$   
 $x_2 \equiv x$   $\frac{dM}{dx} = V$ 

 $A_{c}$ 



#### Kinematics:

$$
\gamma_{32} = u_{3,2} - \omega_1^c
$$
  

$$
k_{12} = \omega_{1,2}^c \equiv \kappa \quad \text{(curvature)}
$$

Constitutive law:

$$
\tau_{32} = G\gamma_{32}
$$
  
\n
$$
m_{12} = Ck_{12}
$$
\n
$$
M = -C^{\star}\kappa, \quad C^{\star} \equiv CA_c \equiv EI
$$

# **When to use Cosserat?**

- Size of the microstructure is important (compared to the characteristic wave length of the loading)
- Presence of internal lengths
- There is no scale separation
- Important stress gradients (compared to the size of the microstructure)

-> Shear bands, high-frequency phenomena, THMC couplings

# **Upscaling to Cosserat continuum?**

The target is to derive the CC constitutive law based on the properties of the microstructure.

Not trivial task as **classical asymptotic homogenization approaches are hardly applicable** (no scale separation).

Equivalent continuum approach:



[Forest et al. 1998, Bardet & Vardoulakis, 2001, Chang & Kung 2005, Gerolymatou, 2011, Godio et al., 2015,16, …]

# **How can we describe multiple failure mechanisms with Cosserat continua?**



Upscaling procedure and transfer of internal lengths

[Stefanou et al.,2008, Godio et al. 2014, 2015, 2016a,b, 2017] [Forest et al., 1998, 1999, 2001]

# **Cosserat for masonry**



*discrete elements model Cosserat model*



#### Masonry is a great toy model for developing higher order theories

# **Cosserat for EQ faults**



[courtesy: Kramer 1996]



#### **Damaged zone**

thickness: from ~10 m to ~1 km

#### **Gouge**

composed of very fine crushed particles, where the slip is localized thickness: from ~1 μm to ~10 mm

A fault zone is modelled as an infinite layer under shear:

Momentum balance equations:

$$
\tau_{ij,j} - \rho \frac{\partial^2 U_i}{\partial t^2} = 0
$$

$$
\mu_{ij,j} - e_{ijk}\tau_{jk} - \rho I \frac{\partial^2 \omega_i^c}{\partial t^2} = 0
$$

Elasto-plastic constitutive equation:

 $\dot{\tau'}_{ij} = C^{ep}_{ijkl} \dot{\gamma}_{kl} + D^{ep}_{ijkl} \dot{\kappa}_{kl} + E^{ep}_{ijkl} \dot{T} \delta_{kl}$  $\dot{\mu}_{ij} = M_{ijkl}^{ep} \dot{\kappa}_{kl} + L_{ijkl}^{ep} \dot{\gamma}_{kl} + N_{ijkl}^{ep} \dot{T} \delta_{kl}$ 

Terzaghi effective stress:  $\tau'_{ij} = \tau_{ij} + p \delta_{ij}$ 

#### Energy balance equation:

$$
\frac{\partial T}{\partial t} - c_{th} T_{,ii} = \frac{1}{\rho C} \underbrace{(\sigma_{ij} \dot{\varepsilon}_{ij}^p + \mu_{ij} \dot{\kappa}_{ij}^p)}_{\text{Plastic work}}
$$

#### Mass balance equation:



I.Stefanou, Oct18

# **Numerical results**

## Fast rate (1 m/s)



I.Stefanou, Oct18 52

# **Grain size evolution based on Breakage Mechanics**



# Micromorphic (generalized) Continua

# **The starting hypotheses**



Ansatz:  $V'_{i} = V_{i} + \chi_{ij}x'_{j} + \chi_{ijk}x'_{j}x'_{k} + \chi_{ijkl}x'_{j}x'_{k}x'_{l} + ...$  $\widetilde{p}^{(ext,t)} = t_i \widetilde{v}_i + \mu_{ij} \widetilde{\chi}_{ij} + \mu_{ijk} \widetilde{\chi}_{ijk} + \dots$  $\widetilde{p}^{(ext,f)} = f_i \widetilde{v}_i + \psi_{ij} \widetilde{\chi}_{ij} + \psi_{ijk} \widetilde{\chi}_{ijk} + \dots$  $\widetilde{p}^{int} = \tau_{ij} \widetilde{V}_{i,j}$  $-(s_{ij}\widetilde{\chi}_{ij}+s_{ijk}\widetilde{\chi}_{ijk}+\ldots)$  $+(\nu_{ijk}\tilde{\kappa}_{ijk}+\nu_{ijkl}\tilde{\kappa}_{ijkl}...)$ 

[Germain, 1973, Mindlin, 1964 Eringen, 1999, ...] [Source The Contract of the Contract of the Contract of the Co

# **Classification**



# Applying PVP: strong form...

$$
\tau_{ij,j} + f_i = 0, \qquad t_i = \tau_{ij} n_j
$$
  

$$
\nu_{ijk,k} + s_{ij} + \psi_{ij} = 0, \qquad \mu_{ij} = \nu_{ijk} n_k
$$
  

$$
\nu_{ijkl,l} + s_{ijk} + \psi_{ijk} = 0, \qquad \mu_{ijk} = \nu_{ijkl} n_l
$$

 $\cdots$ 

# **Exercise #4**

-> *From the general micromorphic equations and by setting:*

$$
\chi_{ij} = -\epsilon_{ijk}\omega_k^c, k_{ij} = \omega_{i,j}^c, s_{ij} = -\frac{1}{2}\epsilon_{ijk}s_k, \mu_{ij} = -\frac{1}{2}\epsilon_{ijk}\mu_k,
$$
  

$$
\nu_{ijk} = -\frac{1}{2}\epsilon_{ijl}m_{lk}, \psi_{ij} = -\frac{1}{2}\epsilon_{ijk}\psi_k, \tau_{ij} \equiv \sigma_{ij} + s_{ij}
$$

*retrieve the strong form of equilibrium equations for Cosserat.*

(hint:  $\epsilon_{ijp} \epsilon_{ijk} = 2 \delta_{pk}$ )

# Finite Elements

# **Example:** *Simple shear with Cauchy continuum*



$$
\text{PVP:} \qquad \mathcal{P} = 0, \forall \widetilde{v}_i \iff \int_D \widetilde{p}^{(int)} dV = \int_{\partial D} \widetilde{p}^{(ext)} dS, \forall \widetilde{v}_i
$$

# **Example:** *Simple shear with Cauchy continuum*

$$
\widetilde{p}^{(int)} = \sigma_{22}\widetilde{v}_{2,2} + \sigma_{12}\widetilde{v}_{1,2} \qquad \sigma_{22} = M_{el}u_{2,2}
$$
  

$$
\widetilde{p}^{(ext)} = \sigma_n \widetilde{v}_2 + \tau \widetilde{v}_1 \qquad \sigma_{12} = G_{el}u_{1,2}
$$

$$
\widetilde{\mathcal{P}} = 0, \forall \widetilde{v}_i \iff \int_D \widetilde{p}^{(int)} dV = \int_{\partial D} \widetilde{p}^{(ext)} dS, \forall \widetilde{v}_i
$$

#### *Code snippet:*

```
# Define internal virtual power
           Pint=(M_e = kDx(u[0], 0) *Dx(v[0], 0) + \#sigma_2^2 * v_2^2G_el*Dx(u[1],0)*Dx(v[1],0) #sigma_12*v_1,2
             )*dxFENICS
           # Define external virtual power
D \cap O \cup E \subset D Pext=dot(ti, v)*ds
```
# Solve the problem (does the FE formulation, matrix assembly and linear solve)  $solve(Pint == Pext, sol, bc)$ 

# **Summary**

- Basic ideas and intuition behind the Principle of Virtual Powers
- **Equivalence between equilibrium and PVP**
- Use in discrete systems
- Use in classical continuum mechanics
- Use in generalized continua
- Use in upscaling where scale separation is no more valid
- Use in Finite Elements (to be continued...)
- Do the exercises…

# **References to start from…**

Principle of Virtual Powers and Thermodynamics

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#### Generalized continua

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*Now that you fell asleep… Soils and rocks… Multiphysical couplings… ??? Energy and Thank you for your attention! balance -> Force*