

ALERT Geomaterials Doctoral School 2018

# The Principle of Virtual Powers definition and use

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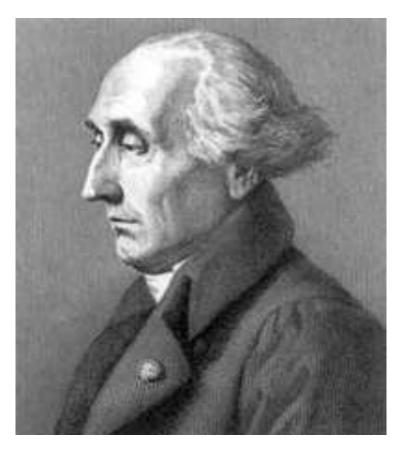
### **Objectives**

- Understand the Principle of Virtual Powers
- Use it in simple problems of statics
- Use it in continuum mechanics
- Use it as a starting point for Finite Element formulations

#### **Prerequisites**

- Tensor calculus
- Patience and a nice problem to solve

# The Principle of Virtual Powers



*Giuseppe Luigi Lagrangia, 1736-1813 (Joseph-Louis Lagrange)* 

# MÉCHANIQUE

#### ANALITIQUE;

Par M. DE LA GRANGE, de l'Académie des Sciences de Paris,

de celles de Berlin, de Pétersbourg, de Turin, &c.



#### A PARIS,

Chez LA VEUVE DESAINT, Libraire, rue du Foin S. Jacques.





## MÉCHANIQUE ANALITIQUE.

PREMIERE PARTIE.

LA STATIQUE.

#### SECTION PREMIERE.

Sur les différens Principes de la Statique.

L'A Statique est la science de l'équilibre des forces. On entend en général par *force* ou *puissance* la cause, quelle qu'elle soit, qui imprime ou tend à imprimer du mouvement au corps auquel on la suppose appliquée; & c'est aussi par la quantité du mouvement imprimé, ou prêt à imprimer, que la force ou puissance doit s'estimer.



#### MÉCHANIQUE ANALITIQUE.

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- **Statics** is the science of the **equilibrium of forces**
- With *force* (or *power*) we mean the *cause of mouvement*
- It is through the quantity of mouvement that the force can be quantified

#### The Principle of Virtual Velocities (or Powers)

Si un fystême quelconque de tant de corps ou points que t'on veut tirés, chacun par des puissances quelconques, est en équilibre, & qu'on donne à ce systême un petit mouvement quelconque, en vertu duquel chaque point parcoure un espace infiniment petit qui exprimera sa vitesse virtuelle; la somme des puissances, multipliées chacune par l'espace que le point où elle est appliquée, parcourt suivant la direction de cette même puissance, sera toujours égale à zero, en regardant comme positifs les petits espaces parcourus dans le sens des puissances, & comme négatifs les espaces parcourus dans un sens opposé.

### **Original statement of PVP in English**

If [any system of bodies or points as we want, is acted upon by any system of forces,] is in equilibrium, and we give to this system any small motion, then [by virtue of the fact that each point travels an infinitesimally small space that expresses its virtual velocity,] the sum of each force multiplied by the space that the point [where it is applied] travels [along the direction of the same power,] will always be equal to zero, [regarding as positive the small distances followed in the

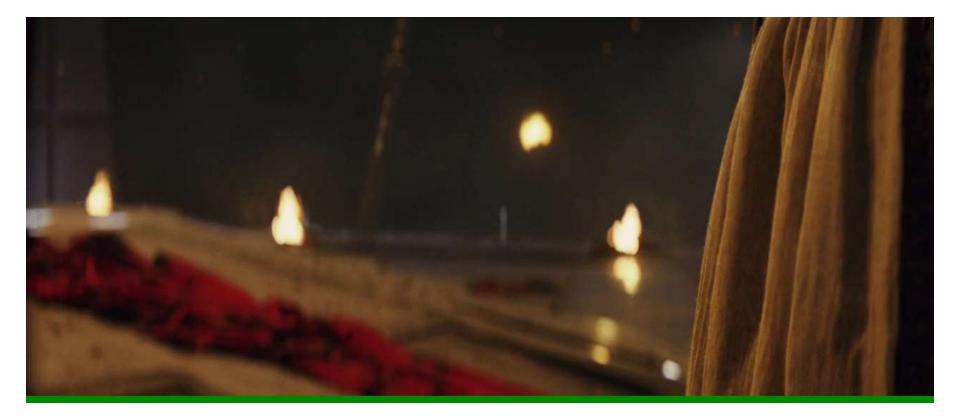
direction of the powers and as negative those travelled in the opposite direction.]

The body will be in equilibrium *if, and only if,* 

the power generated by the forces acting on it, is zero under **any possible (= virtual) velocity** of the body.

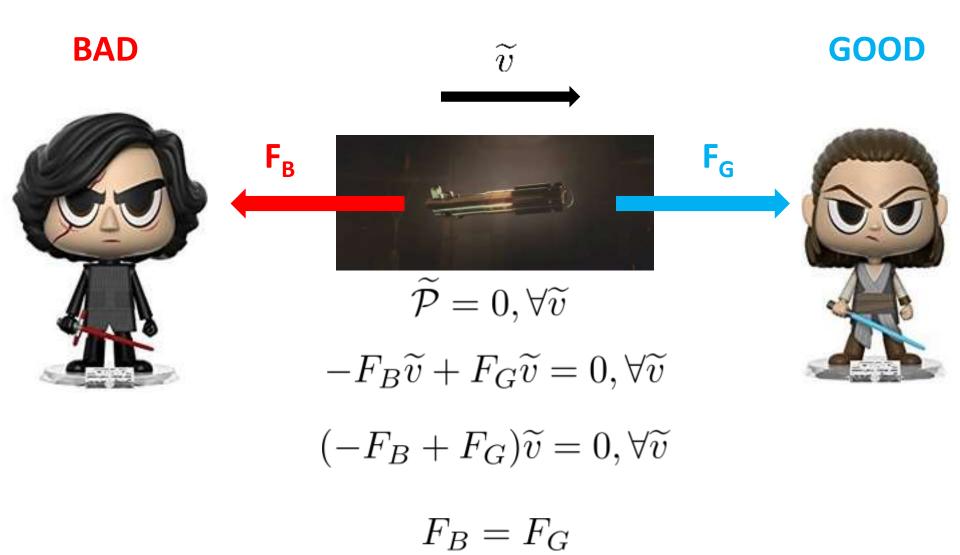
#### Lagrange was far ahead of his time...

A long time ago, in a galaxy far, far away....

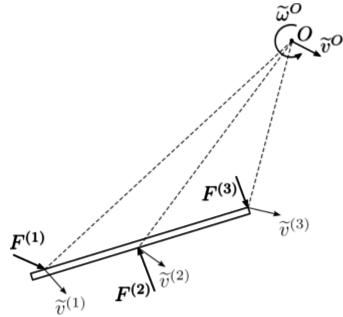


[courtesy: The last Jedi, film, 2018]

### The scientific problem:



#### **PVP** applied to a single body



equilibrium  $\iff \sum_{i} \widetilde{\mathcal{P}}^{(i)} = 0, \forall \widetilde{\boldsymbol{v}}^{(i)}$  $\sum_{i} \boldsymbol{F}^{(i)} \cdot \widetilde{\boldsymbol{v}}^{(i)} = 0, \forall \widetilde{\boldsymbol{v}}^{(i)}$ Rigid body:  $\widetilde{m{v}}^{(i)} = \widetilde{m{v}}^O + \widetilde{m{\omega}}^O imes m{r}^{(O,i)}$  $\left(\sum_{i} \boldsymbol{F}^{(i)}\right) \cdot \widetilde{\boldsymbol{v}}^{O} + \left(\sum_{i} \boldsymbol{r}^{(O,i)} \times \boldsymbol{F}^{(i)}\right) \cdot \widetilde{\boldsymbol{\omega}}^{O} = 0, \forall \widetilde{\boldsymbol{v}}^{O}, \widetilde{\boldsymbol{\omega}}^{O}$  $\sum_{i} \boldsymbol{F}^{(i)} = 0$  $orall \widetilde{v}^O, \widetilde{\omega}^O \implies$ 

 $\sum_i \boldsymbol{M}^{(O,i)} = 0$ 

#### **PVP and 'Newton' equivalence**

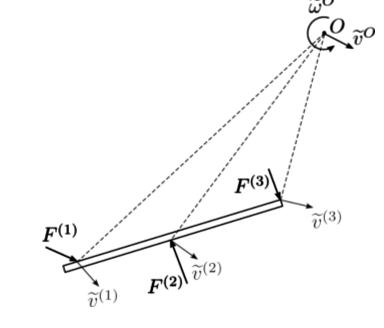
equilibrium 
$$\iff \sum_{i} \mathbf{F}^{(i)} = 0$$
  
$$\sum_{i} \mathbf{M}^{(O,i)} = 0$$

multiplying by any  $\widetilde{v}^O, \widetilde{\omega}^O$  and adding:

$$\left(\sum_{i} \boldsymbol{F}^{(i)}\right) \cdot \widetilde{\boldsymbol{v}}^{O} + \left(\sum_{i} \boldsymbol{r}^{(O,i)} \times \boldsymbol{F}^{(i)}\right) \cdot \widetilde{\boldsymbol{\omega}}^{O} = 0, \forall \widetilde{\boldsymbol{v}}^{O}, \widetilde{\boldsymbol{\omega}}^{O}$$

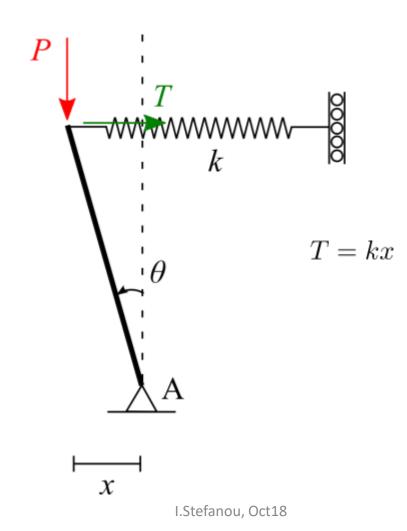
Rigid body:  $\widetilde{v}^{(i)} = \widetilde{v}^O + \widetilde{\omega}^O \times r^{(O,i)}$ 

$$\sum_{i} \boldsymbol{F}^{(i)} \cdot \widetilde{\boldsymbol{v}}^{(i)} = 0, \forall \widetilde{\boldsymbol{v}}^{(i)} \iff \sum_{i} \widetilde{\mathcal{P}}^{(i)} = 0, \forall \widetilde{\boldsymbol{v}}^{(i)}$$

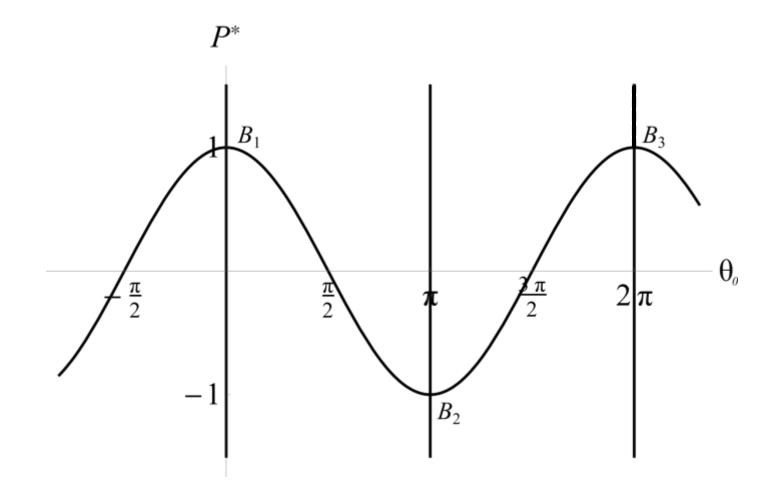


#### Exercise #1

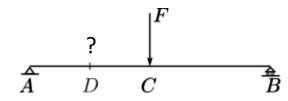
-> Using the PPV find all the equilibrium points (angles ϑ) of the system:

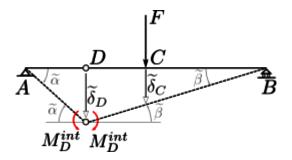


#### Answer #1



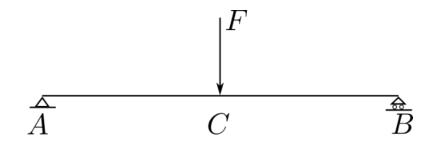
#### PVP applied to a system of bodies





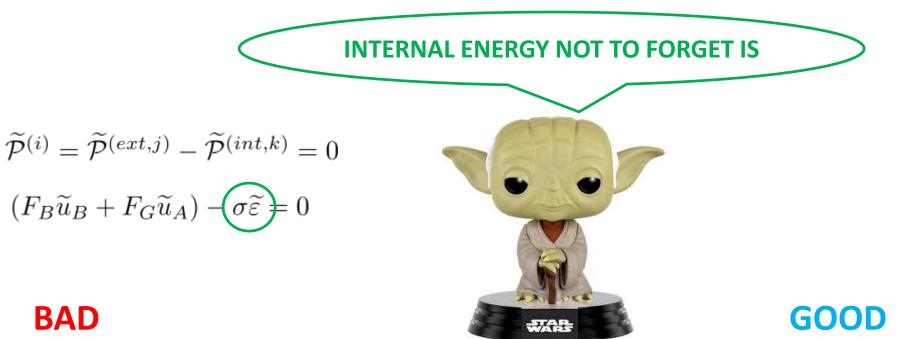
#### Exercise #2

-> Using the PPV find the internal moment at point C:

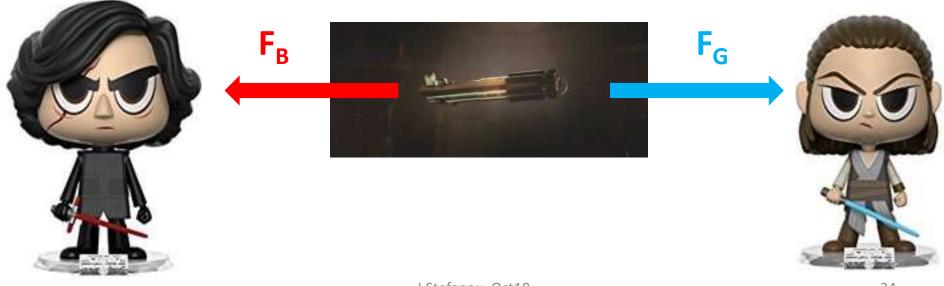


#### Answer #2

# $M_D^{int} = \frac{FL}{4}$



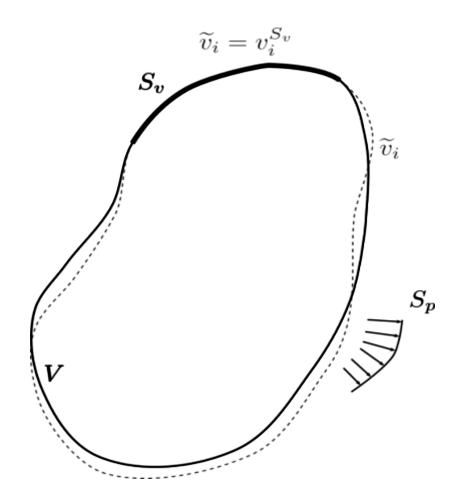
**BAD** 

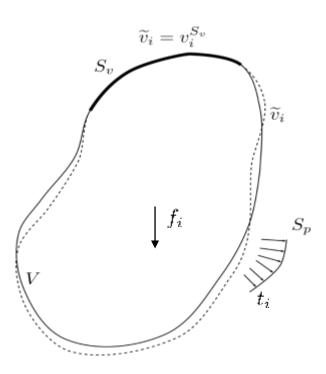




# **Continuum Mechanics**

### **PVP and continuum mechanics**





#### Definitions

internal deformation enery density:  $\tilde{p}^{(int)} = \sigma_{ij} \tilde{\varepsilon}_{ij}$ 

$$\widetilde{\varepsilon}_{ij} = \frac{1}{2} \left( \widetilde{v}_{i,j} + \widetilde{v}_{j,i} \right), \quad \sigma_{ij} = \sigma_{ji}$$

external enery density:  $\tilde{p}^{(ext,t)} = t_i \tilde{v}_i$ 

$$\widetilde{p}^{(ext,f)} = f_i \widetilde{v}_i$$

$$\int_{V} \sigma_{ij} \widetilde{\varepsilon}_{ij} dV - \int_{V} f_i \widetilde{v}_i dV - \int_{\partial V} t_i \widetilde{v}_i dS = 0, \forall \widetilde{v}_i$$

$$\int_{V} \sigma_{ij} \widetilde{\varepsilon}_{ij} dV = \int_{V} \sigma_{ij} \widetilde{v}_{i,j} dV = \int_{V} (\sigma_{ij} \widetilde{v}_{i})_{,j} dV - \int_{V} \sigma_{ij,j} \widetilde{v}_{i} dV$$

Using the divergence theorem:

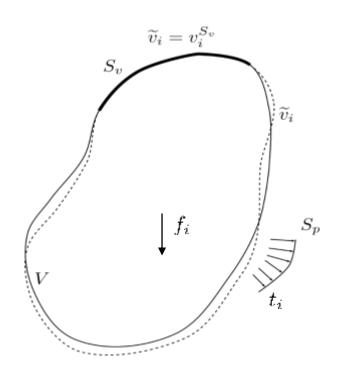
$$\int_{V} (\sigma_{ij} \widetilde{v}_i)_{,j} dV = \int_{\partial V} (\sigma_{ij} \widetilde{v}_i) n_j dS = \int_{\partial V} (\sigma_{ij} n_j) \widetilde{v}_i dS$$

Replacing:  $\int_{V} (\sigma_{ij,j} + f_i) \, \widetilde{v}_i dV + \int_{\partial V} (t_i - \sigma_{ij} n_j) \, \widetilde{v}_i dS = 0, \forall \widetilde{v}_i$ 

$$\int_{V} \left(\sigma_{ij,j} + f_{i}\right) \widetilde{v}_{i} dV + \int_{\partial V} \left(t_{i} - \sigma_{ij} n_{j}\right) \widetilde{v}_{i} dS = 0, \forall \widetilde{v}_{i}$$

$$\begin{array}{l} \forall V \\ \Leftrightarrow \end{array} \begin{cases} \sigma_{ij,j} + f_i = 0 \\ \\ t_i = \sigma_{ij} n_j \end{array}$$

Which are the classical equilibrium equations in continuum mechanics -> Strong form



#### Linear & angular momentum balance principles

• 
$$\int_{\partial V} t_i dS + \int_V f_i dV = 0$$
  
with  $t_i = \sigma_{ij} n_j$   
• 
$$\int_{\partial V} \epsilon_{ijk} t_j x_k dS + \int_{\partial V} \epsilon_{ijk} f_j x_k dV = 0$$
  

$$\int_{\partial V} t dS + \int_V f dV = \int_{\partial V} \vec{t} dS + \int_V \vec{f} dV$$
  

$$\int_{\partial V} t \times x dS = \int_{\partial V} \vec{t} \times \vec{x} dS$$

$$\int_{\partial V} t_i dS + \int_V f_i dV = 0$$

 $t_i = \sigma_{ij} n_j$ 

Using the divergence theorem:

$$\int_{\partial V} t_i dS = \int_{\partial V} \sigma_{ij} n_j dS = \int_V \sigma_{ij,j} dV$$

Replacing:

$$\int_{V} (\sigma_{ij,j} + f_i) dV = 0$$

$$\int_{\partial V} \epsilon_{ijk} t_j x_k dS + \int_V \epsilon_{ijk} f_j x_k dV = 0$$

 $t_i = \sigma_{ij} n_j$ 

Using the divergence theorem:

$$\int_{\partial V} \epsilon_{ijk} t_j x_k dS = \int_{\partial V} \epsilon_{ijk} \sigma_{jp} x_k n_p dS = \int_V (\epsilon_{ijk} \sigma_{jp} x_k)_{,p} dV =$$

$$= \int_{V} (\epsilon_{ijk} \sigma_{jp} x_k)_{,p} dV = \int_{V} \epsilon_{ijk} \sigma_{jp,p} x_k dV + \int_{V} \epsilon_{ijk} \sigma_{jp} x_{k,p} dV =$$

$$= \int_{V} \epsilon_{ijk} (-f_j) x_k dV + \int_{V} \epsilon_{ijk} \sigma_{jp} \delta_{kp} dV$$

$$= -\int_V \epsilon_{ijk} f_j x_k dV + \int_V \epsilon_{ijk} \sigma_{jk} dV$$

#### Replacing:

$$\int_{\partial V} \epsilon_{ijk} t_j x_k dS + \int_V \epsilon_{ijk} f_j x_k dV = \int_V \epsilon_{ijk} \sigma_{jk} dV = 0$$

$$\int_V \epsilon_{ijk} \sigma_{jk} dV = 0$$

$$\sigma_{jk} = \sigma_{kj}$$

Starting point:

- Virtual (generalized) displacements
- Virtual powers (external & internal)

Less intuition

Prone to the Galerkin Method (Finite Elements)

#### **Momentum Balance**

Starting point:

• (Generalized) stresses

More intuition

#### **Principle of Virtual Powers**

or

#### **Principle linear & angular momentum balance ?**

#### It is a matter of principles!

## **Cosserat Continuum**

Lecture Notes in Applied and Computational Mechanics 87

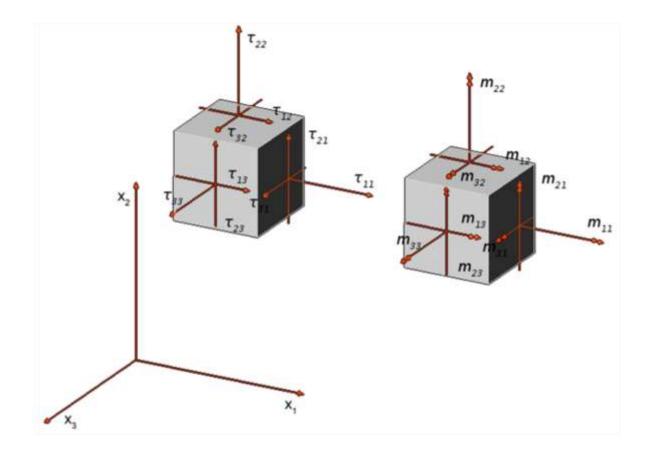
#### Ioannis Vardoulakis

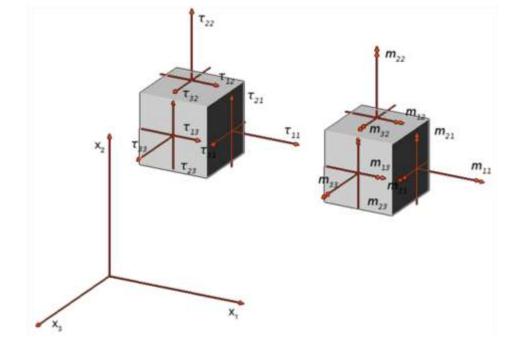
# Cosserat Continuum Mechanics

With Applications to Granular Media



## From the MB point of view

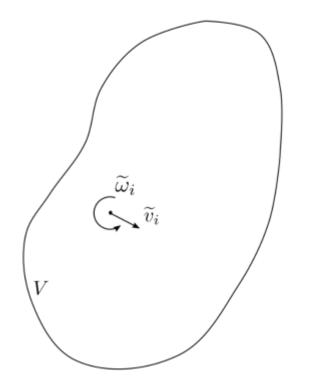




# Equilibrium on infinitesimal volume & Cauchy tetrahedron:

$$\tau_{ij,j} + f_i = 0, \qquad t_i = \tau_{ij} n_j$$
$$m_{ij,j} - \epsilon_{ijk} \tau_{jk} + \psi_i = 0, \qquad \mu_i = m_{ij} n_j$$

## **Energetic approach: PVP**



$$\widetilde{p}^{(ext,f)} = f_i \widetilde{v}_i + \psi_i \widetilde{\omega}_i^c$$
$$\widetilde{p}^{(ext,t)} = t_i \widetilde{v}_i + \mu_i \widetilde{\omega}_i^c$$
$$\widetilde{p}^{int} = \tau_{ij} \widetilde{\gamma}_{ij} + m_{ij} \widetilde{k}_{ij}$$
$$\gamma_{ij} = u_{i,j} + \epsilon_{ijk} \omega_k^c$$
$$k_{ij} = \omega_{i,j}^c$$

#### **Exercise #3**

-> Retrieve the strong form of equilibrium equations for Cosserat:

$$\widetilde{p}^{(ext,f)} = f_i \widetilde{v}_i + \psi_i \widetilde{\omega}_i^c$$
$$\widetilde{p}^{(ext,t)} = t_i \widetilde{v}_i + \mu_i \widetilde{\omega}_i^c$$
$$\widetilde{p}^{int} = \tau_{ij} \widetilde{\gamma}_{ij} + m_{ij} \widetilde{k}_{ij}$$
$$\gamma_{ij} = u_{i,j} + \epsilon_{ijk} \omega_k^c$$
$$k_{ij} = \omega_{i,j}^c$$

$$\tau_{ij,j} + f_i = 0$$
  

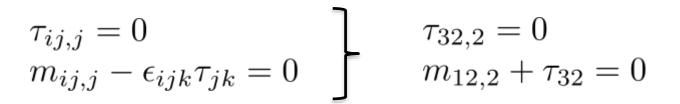
$$m_{ij,j} - \epsilon_{ijk}\tau_{jk} + \psi_i = 0$$
  

$$t_i = \tau_{ij}n_j$$
  

$$\mu_i = m_{ij}n_j$$

### **1D Cosserat: The Timoshenko beam**

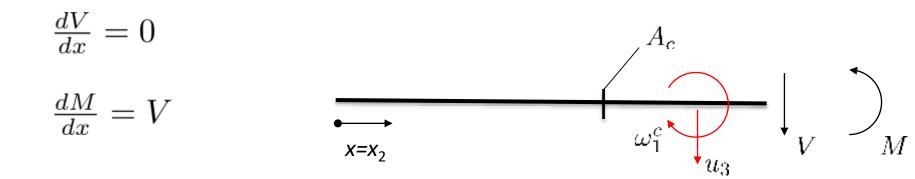
Consider a narrow infinite strip of Cosserat continuum in the  $x_2$ -direction, invariant in the  $x_1$ - and  $x_3$ -directions:



 $X_2$ 

$$V \equiv \tau_{32} A_c \qquad \qquad \frac{dV}{dx} = 0$$
  
Call:  $M \equiv -m_{12} A_c \longrightarrow \qquad \qquad \frac{dM}{dx} = V_{_{43}}$ 

 $A_c$ 



#### Kinematics:

$$\gamma_{32}=u_{3,2}-\omega_1^c$$
  
 $k_{12}=\omega_{1,2}^c\equiv\kappa$  (curvature)

Constitutive law:

$$\tau_{32} = G\gamma_{32} \longrightarrow \begin{cases} V = G^* \frac{du_3}{dx}, & G^* \equiv GA_c \\ M = -C^*\kappa, & C^* \equiv CA_c \equiv EI \end{cases}$$

## When to use Cosserat?

- Size of the microstructure is important (compared to the characteristic wave length of the loading)
- Presence of internal lengths
- There is no scale separation
- Important stress gradients (compared to the size of the microstructure)

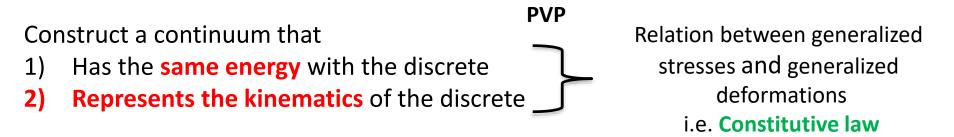
-> Shear bands, high-frequency phenomena, THMC couplings

## **Upscaling to Cosserat continuum?**

The target is to derive the CC constitutive law based on the properties of the microstructure.

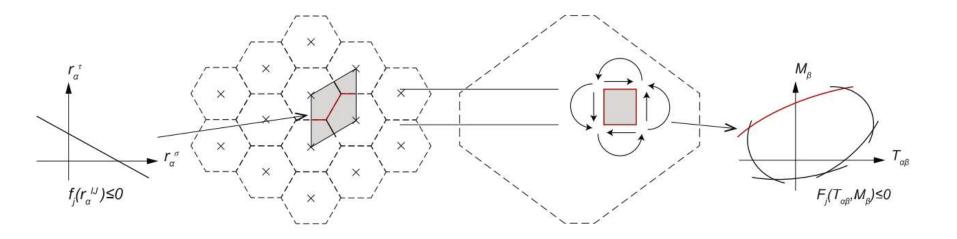
Not trivial task as **classical asymptotic homogenization approaches are hardly applicable** (no scale separation).

Equivalent continuum approach:



[Forest et al. 1998, Bardet & Vardoulakis, 2001, Chang & Kung 2005, Gerolymatou, 2011, Godio et al., 2015,16, ...]

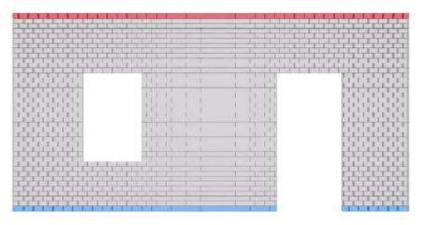
# How can we describe multiple failure mechanisms with Cosserat continua?



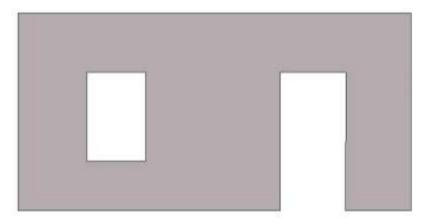
Upscaling procedure and transfer of internal lengths

[Forest et al., 1998, 1999, 2001] [Stefanou et al.,2008, Godio et al. 2014, 2015, 2016a,b, 2017]

#### **Cosserat for masonry**



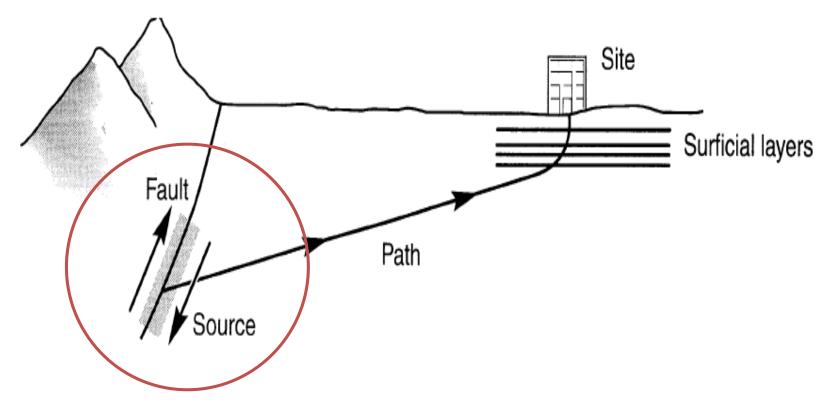
discrete elements model



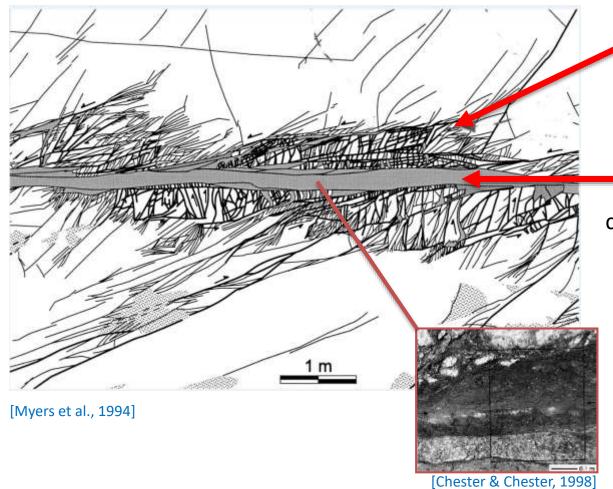
Cosserat model

#### Masonry is a great toy model for developing higher order theories

#### **Cosserat for EQ faults**



[courtesy: Kramer 1996]



#### Damaged zone

thickness: from ~10 m to ~1 km

#### Gouge

composed of very fine crushed particles, where the slip is localized thickness: from ~1 μm to ~10 mm A fault zone is modelled as an infinite layer under shear:

Momentum balance equations:

$$\tau_{ij,j} - \rho \frac{\partial^2 U_i}{\partial t^2} = 0$$
$$\mu_{ij,j} - e_{ijk} \tau_{jk} - \rho I \frac{\partial^2 \omega_i^c}{\partial t^2} = 0$$

Elasto-plastic constitutive equation:

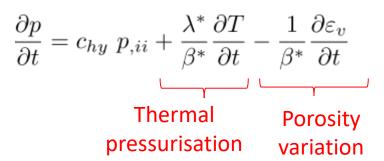
 $\dot{\tau'}_{ij} = C^{ep}_{ijkl} \dot{\gamma}_{kl} + D^{ep}_{ijkl} \dot{\kappa}_{kl} + E^{ep}_{ijkl} \dot{T} \delta_{kl}$  $\dot{\mu}_{ij} = M^{ep}_{ijkl} \dot{\kappa}_{kl} + L^{ep}_{ijkl} \dot{\gamma}_{kl} + N^{ep}_{ijkl} \dot{T} \delta_{kl}$ 

Terzaghi effective stress:  $au_{ij}' = au_{ij} + p \,\, \delta_{ij}$ 

Energy balance equation:

$$\frac{\partial T}{\partial t} - c_{th}T_{,ii} = \frac{1}{\rho C} (\sigma_{ij}\dot{\varepsilon}^{p}_{ij} + \mu_{ij}\dot{\kappa}^{p}_{ij})$$
Plastic work

#### Mass balance equation:

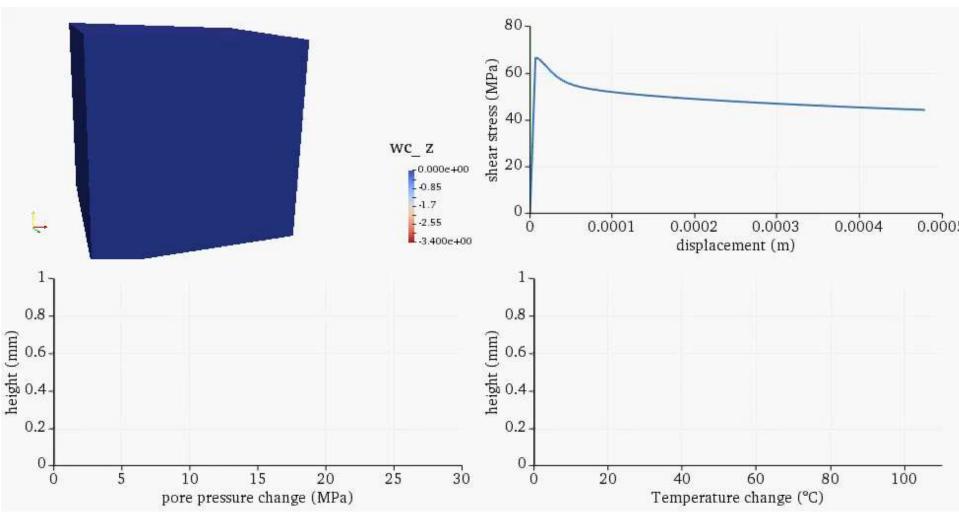


[Rattez et al. 2018a,b,c]

I.Stefanou, Oct18

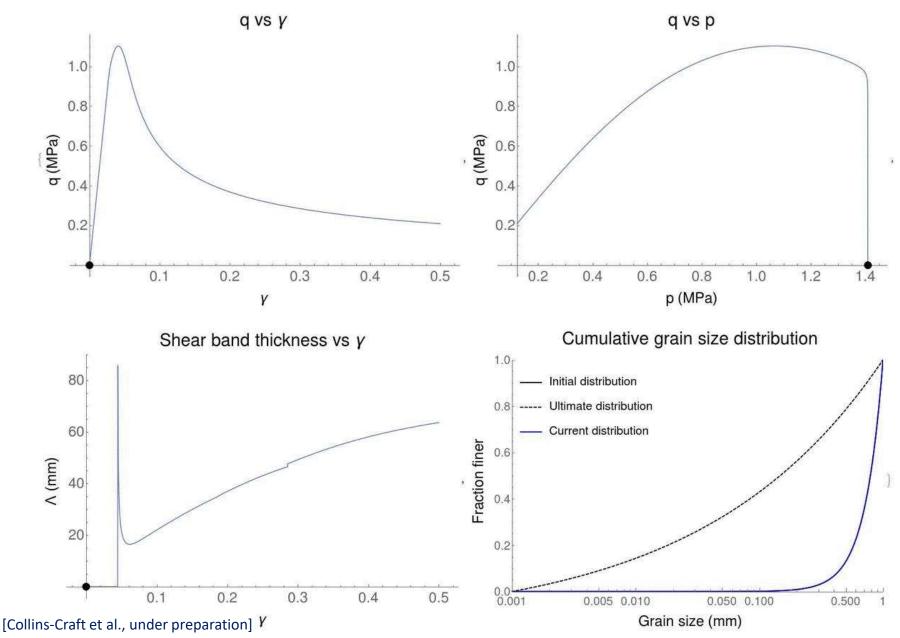
#### **Numerical results**

#### Fast rate (1 m/s)



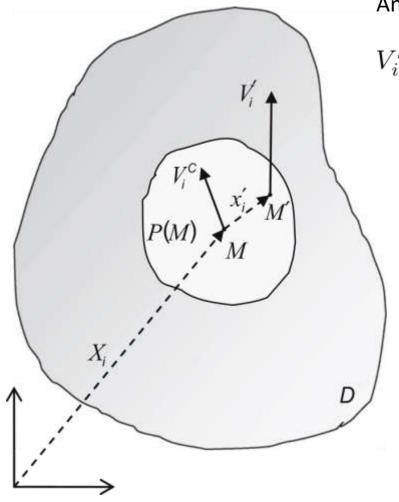
I.Stefanou, Oct18

#### **Grain size evolution based on Breakage Mechanics**



# Micromorphic (generalized) Continua

### The starting hypotheses



Ansatz:

$$X'_{i} = V_{i} + \chi_{ij}x'_{j} + \chi_{ijk}x'_{j}x'_{k} + \chi_{ijkl}x'_{j}x'_{k}x'_{l} + \dots$$

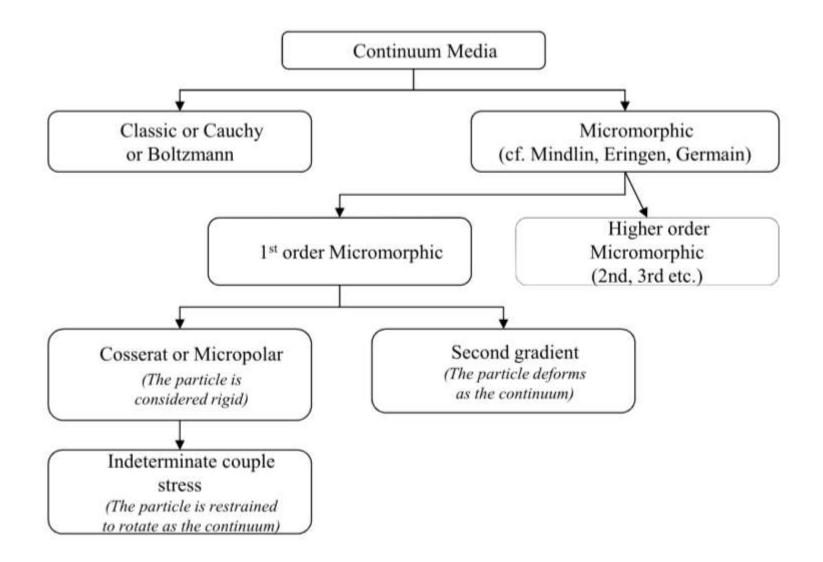
$$\widetilde{p}^{(ext,t)} = t_i \widetilde{v}_i + \mu_{ij} \widetilde{\chi}_{ij} + \mu_{ijk} \widetilde{\chi}_{ijk} + \dots$$

$$\widetilde{p}^{(ext,f)} = f_i \widetilde{v}_i + \psi_{ij} \widetilde{\chi}_{ij} + \psi_{ijk} \widetilde{\chi}_{ijk} + \dots$$

$$\widetilde{p}^{int} = \tau_{ij} \widetilde{V}_{i,j} - (s_{ij} \widetilde{\chi}_{ij} + s_{ijk} \widetilde{\chi}_{ijk} + \ldots) + (\nu_{ijk} \widetilde{\kappa}_{ijk} + \nu_{ijkl} \widetilde{\kappa}_{ijkl} \ldots)$$

[Germain, 1973, Mindlin, 1964 Eringen, 1999, ...]

## Classification



### **Applying PVP: strong form...**

$$\tau_{ij,j} + f_i = 0, \qquad t_i = \tau_{ij} n_j$$
$$\nu_{ijk,k} + s_{ij} + \psi_{ij} = 0, \qquad \mu_{ij} = \nu_{ijk} n_k$$
$$\nu_{ijkl,l} + s_{ijk} + \psi_{ijk} = 0, \qquad \mu_{ijk} = \nu_{ijkl} n_l$$

. . .

#### **Exercise #4**

-> From the general micromorphic equations and by setting:

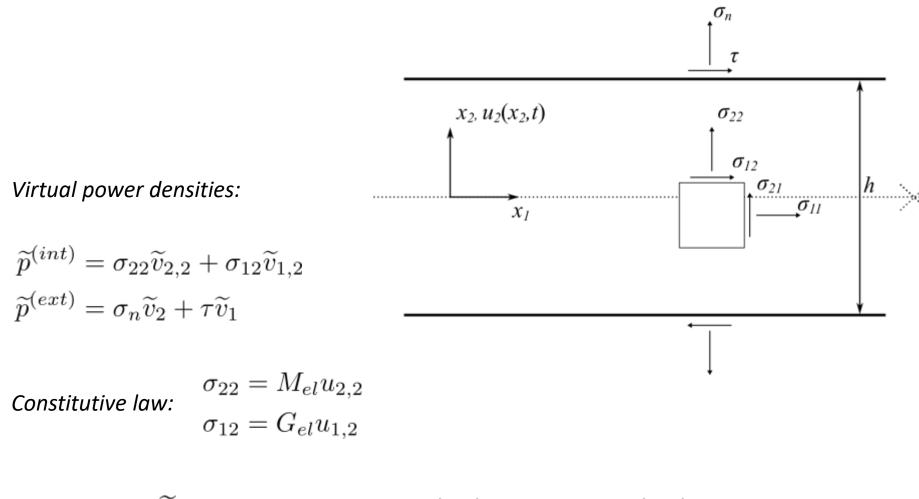
$$\chi_{ij} = -\epsilon_{ijk}\omega_k^c, \, k_{ij} = \omega_{i,j}^c, \, s_{ij} = -\frac{1}{2}\epsilon_{ijk}s_k, \, \mu_{ij} = -\frac{1}{2}\epsilon_{ijk}\mu_k, \\ \nu_{ijk} = -\frac{1}{2}\epsilon_{ijl}m_{lk}, \, \psi_{ij} = -\frac{1}{2}\epsilon_{ijk}\psi_k, \, \tau_{ij} \equiv \sigma_{ij} + s_{ij}$$

retrieve the strong form of equilibrium equations for Cosserat.

(hint:  $\epsilon_{ijp}\epsilon_{ijk} = 2\delta_{pk}$  )

## **Finite Elements**

### Example: Simple shear with Cauchy continuum



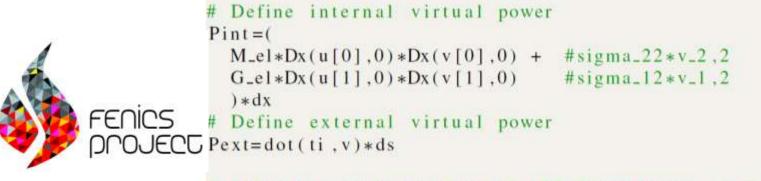
PVP: 
$$\widetilde{\mathcal{P}} = 0, \forall \widetilde{v}_i \iff \int_D \widetilde{p}^{(int)} dV = \int_{\partial D} \widetilde{p}^{(ext)} dS, \forall \widetilde{v}_i$$

### Example: Simple shear with Cauchy continuum

$$\widetilde{p}^{(int)} = \sigma_{22}\widetilde{v}_{2,2} + \sigma_{12}\widetilde{v}_{1,2} \qquad \qquad \sigma_{22} = M_{el}u_{2,2}$$
  
$$\widetilde{p}^{(ext)} = \sigma_n\widetilde{v}_2 + \tau\widetilde{v}_1 \qquad \qquad \sigma_{12} = G_{el}u_{1,2}$$

$$\widetilde{\mathcal{P}} = 0, \forall \widetilde{v}_i \iff \int_D \widetilde{p}^{(int)} dV = \int_{\partial D} \widetilde{p}^{(ext)} dS, \forall \widetilde{v}_i$$

#### *Code snippet:*



# Solve the problem (does the FE formulation, matrix assembly and linear solve) solve(Pint == Pext, sol, bc)

## Summary

- Basic ideas and intuition behind the Principle of Virtual Powers
- Equivalence between equilibrium and PVP
- Use in discrete systems
- Use in classical continuum mechanics
- Use in generalized continua
- Use in upscaling where scale separation is no more valid
- Use in Finite Elements (to be continued...)
- Do the exercises...

## **References to start from...**

Principle of Virtual Powers and Thermodynamics

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Thank you for your attention!